



GM Locomotive Group

M.I. 100 MAINTENANCE INSTRUCTION

*Rev. C

CRANKCASE MAIN BEARING STEEL BORE ALIGNMENT QUALIFICATION

INTRODUCTION

The purpose of this maintenance instruction is to define the dimensions and limits which control the size, shape, and alignment of the 567, 645 and 710 crankcase main bearing bores. The text of the instruction is divided into four main sections as follows:

- I Determining If Line Bore Is Necessary
- II. Preparation For Line Bore
- III. Qualification Of Line Bore.
Granite Table Method
- IV. Engine Rebuild And Load Testing

A flow chart is provided to show the recommended basic sequence to be followed so that this qualification process can be performed efficiently.

NOTE

This Maintenance Instruction does not apply to crankcases manufactured or rebuilt prior to 1972.

PROCEDURE

SECTION I

DETERMINING IF LINE BORE IS REQUIRED

Obvious failures involving the main bearing/crankshaft system will require complete disassembly of the engine and in almost all cases will also require line bore. However, those engines which have been taken

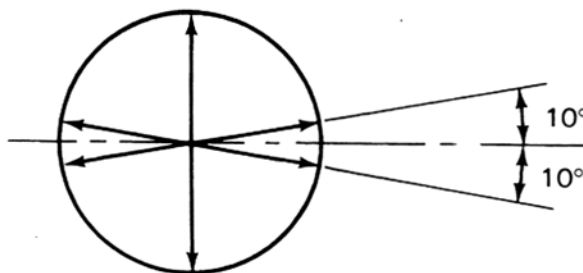
out of service for reasons other than an obvious failure of this system can have the main bearing steel bore alignment requalified with a minimal amount of work.

PRELIMINARY INSPECTION — STEEL BORE DIAMETER MEASUREMENTS

With the engine disassembled and the crankshaft and main bearings removed, reapply the main bearing caps as indicated in Section II, Item C-4. Measure the steel bore diameters as shown in Fig. 1, using bore gauge 8275258 with master 9321276. Two sets of readings are required per each main frame member (MFM) taken 1/2" in from each side (accessory and generator ends of each MFM). The crankcase may be reused without bore remachining if all of the diameter readings are within the limits provided in the appendix, Table 1 and Table 2. If the crankcase does not meet the limits provided, proceed to Section II.

NOTE

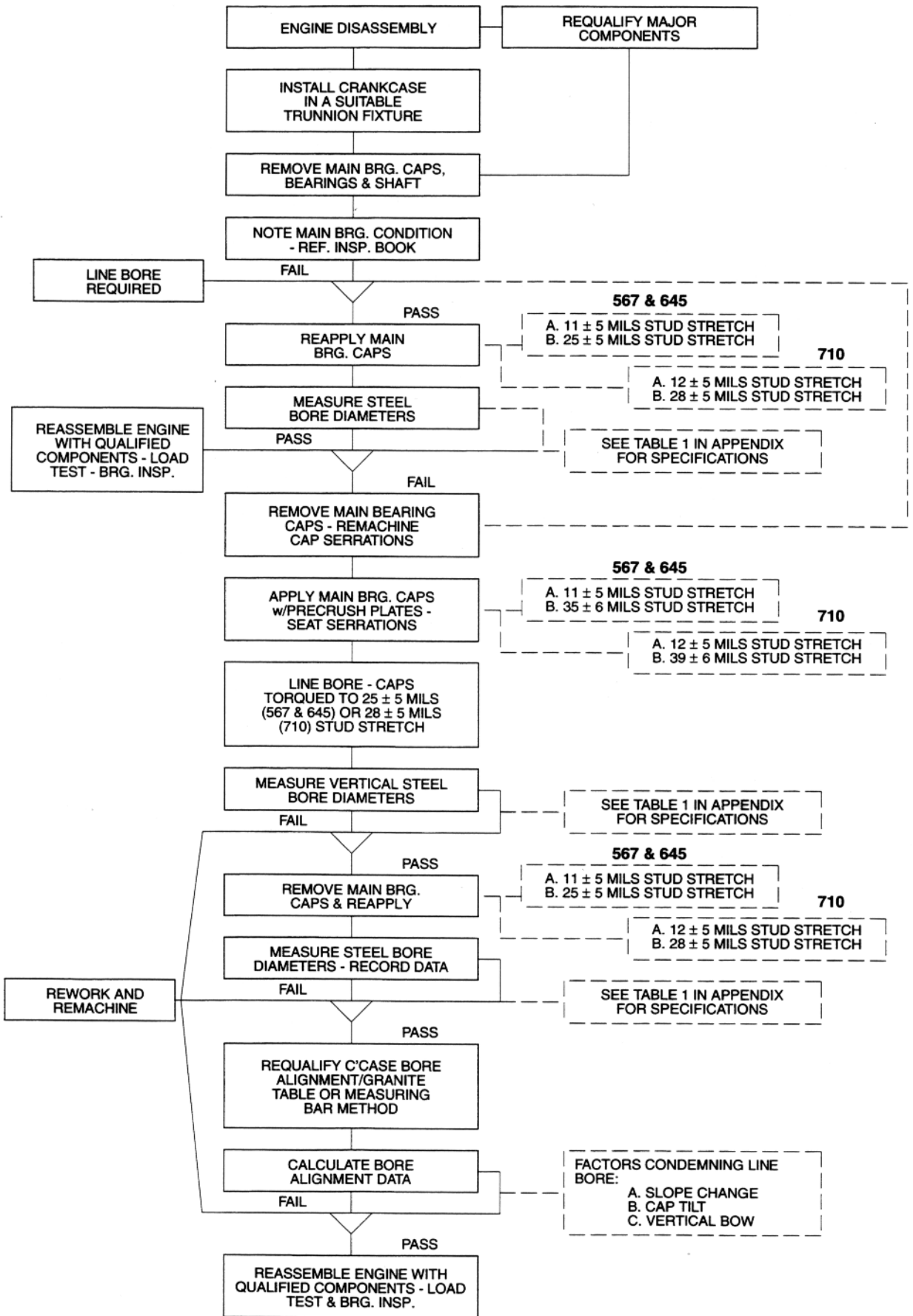
When taking horizontal readings of steel bore diameter, be certain gauge is positioned away from "mud pocket" and main bearing cap "tang slot" in the "A" frame.



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Fig. 1 - Main Bearing Bore Measurement

*This bulletin supersedes previous issues of this number.



SECTION II

PREPARATION FOR LINE BORE

A. QUALIFICATION OF MAIN BEARING NUT TORQUE APPLICATION

Stud stretch measurements should be used as the criteria for monitoring torques. The objective of measuring stud stretch is to determine the actual clamp load of the stud/nut system and thus ensure that the torquing method is effective. Stud stretch is defined as *the overall length change of the stud evaluated as the length when torqued minus the length after the applied torque is removed*. Measurement should be as follows:

Using a 16-inch micrometer or Vernier caliper, measure the length of the stud, being careful not to locate on the small projections found on the ends of each stud. The preferred method is to hold the measuring instrument against the back end of the stud with one hand in such a manner that one finger touches the stud end and locates the point of measurement at approximately 3/8" radially from the side of the stud. The outboard end measurement point can be visually located in line with the inboard end point. Operator ability to use this method, or any other selected method, should be checked by taking repetitive readings. Note, that in order for these readings to agree, they must be taken in the same location on the stud ends to minimize errors due to tapered ends.

The method of torque application substantially affects stud stretch. A qualified air or hydraulically operated wrench is recommended for main bearing nut application. Hand torque application of main bearing nuts may not be reliable and can be expected to give a stud stretch range as high as .016".

To qualify any main bearing nut torque application system, perform the following:

1. Select twelve studs which have been prepared normally for main cap application. (See Section II, Item C.)
2. Torque nuts to 750 ft-lbs.
3. Check and record stud length while torqued.
4. Loosen nuts.
5. Record stud length after removal of torque and determine "stud stretch" by subtracting from the length recorded in Step A 3 above.
6. Reset wrench or system if necessary and repeat test

until the "average" stud stretch is .025" \pm .001" on 567 and 645 engines or .028" \pm .005" on 710 engines. The 12 individual stud stretch measurements should not vary beyond .025" \pm .005".

B. MAIN BEARING CAP AND CRANKCASE SERRATION INSPECTION

Crankcases which require line bore must have the main bearing caps reworked to obtain machining stock and to upgrade the serration quality. Rework is to be as follows:

1. A minimum of .040" (0.64 mm) must be cut off the crest of the serrations, during remachining to ensure adequate cleanup.
2. Cut the nut seat surface to remove any superficial discontinuities. The nut seat surface should be parallel to the serration pitch line to within .010" (0.25 mm).

SERRATION SPECIFICATIONS

NOTE

Serrations should be machined on equipment that has been checked to the specifications listed below. Checking should be done by cutting serrations on a sample piece of metal (coupon) and placing it on a shadow graph (comparator) or similar machine.

1. Serration Spacing.

Measurement from the centerline of No. 2 serration on one bank to the centerline of the No. 2 serration on the opposite bank must be 9.500" \pm .010" (241.30 \pm 0.25 mm) on 567 and 645 engines, and 10.500" \pm .010" (260.35 \pm 0.038 mm) on 710 engines.

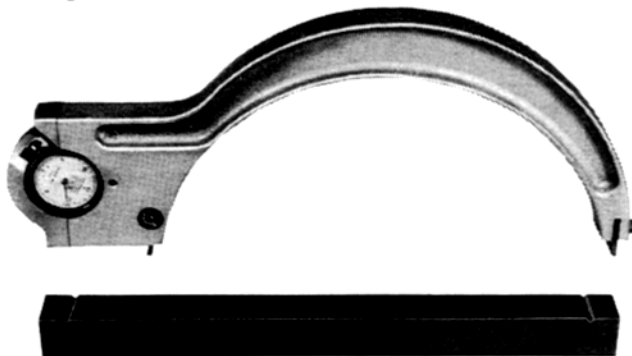
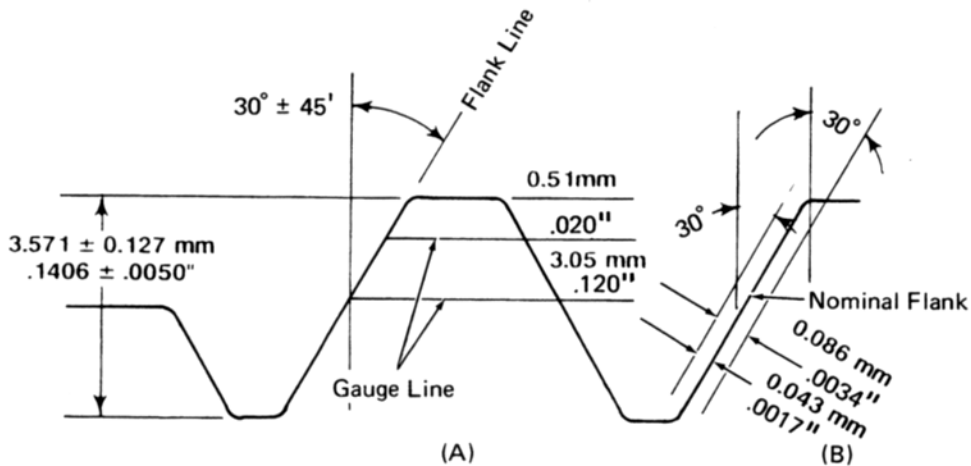


Fig. 2 - Typical Serration Gauge w/Master

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Fig. 3 - Determination Of Serration Quality

2. Flank Angle.

Flank angle is to be measured by locating horizontal gauge lines at .020" (0.51 mm) and .120" (3.05 mm) below the top of the tooth. Draw a line (flank line) through the two points where the gauge lines intersect the tooth profile. This flank line must form a $30^\circ \pm 45'$ angle to the vertical. See (A), Fig. 3.

3. Deviation from nominal serration flank.

.0017" (0.043 mm) - measured perpendicular to nominal flank surface.

The projected outline of the tooth serration must fall entirely between two 30° lines which are .0034" (0.086 mm) apart. See (B), Fig. 3.

C. APPLICATION OF MAIN BEARING CAP AT LINE BORE

NOTE

Stud stretch measurements cannot be made on type "567B" or earlier crankcases. Equivalent hand torques must be used.

1. Apply main bearing cap for pre-crush and line bore using flat plate 8488128 against cap (do not use flat plates on "567B" and earlier crankcases), followed by hardened washer 8412532 and nut 8408684. Apply Texaco Threadtex No. 2303 liberally to stud threads, nut face, and both sides of the washer.

2. Tighten all nuts for pre-crush as follows:

a. Tighten all nuts to produce 11 ± 5 mils stud stretch on 567 and 645 engines (12 ± 5 mils on 710 engines) or 475 N·m (350 ft.-lbs.) Ref. torque.

b. Tighten all nuts to produce 35 ± 6 mils stud stretch on 567 C and later and 645 engines (39 ± 6 mils on 710 engines) or 1491 N·m (1100 ft.-lbs.) (1152 N·m [850 ft.-lbs.] for "567B" and earlier crankcases) Ref. torque.

c. Select four studs at one intermediate cap and measure stud stretch as defined in Section II, Items A2 through A6.

3. Loosen all nuts; measure and record free length on the four studs selected in Step 2c above.

4. Tighten all nuts for line bore as follows:

a. Tighten all nuts to produce 11 ± 5 mils stud stretch on 567 and 645 engines (12 ± 5 mils on 710 engines) or 475 N·m (350 ft.-lbs.) Ref. torque.

b. Tighten all nuts to produce 25 ± 5 mils stud stretch on 567 and 645 engines (28 ± 5 mils on 710 engines) or 1017 N·m (750 ft.-lbs.) Ref. torque all engines, including "567B" and earlier crankcases.

c. Measure and record length of the above four selected studs.

5. Ready for line bore.

D. LINE BORE

The line bore equipment used should be able to produce a bore surface finish of no greater than 200 μ in. (5.08 μ m).

Following line bore, measure the vertical steel bore diameters. All bore measurements are to be taken 1/2" (12.7 mm) in from the generator side and 1/2" in from the accessory side of each MFM. Vertical diameters should be within the 8.251" to 8.249" (209.58 to 209.52 mm) limits for 567 and 645 engines or 9.251" to 9.249" (234.98 to 234.92 mm) limits for 710 engines as shown in Appendix Table 1, or reworked and requalified.

Remove pre-crush flat plate 8488128. **Remove all caps.**

E. RECAP APPLICATION FOR MEASUREMENT OF VERTICAL BORE

The purpose of this **recap** operation is to ensure that proper seating of the MFM and cap serrations was attained prior to line bore.

1. Assemble main bearing caps with standard washers and nuts. Apply Texaco Threadtex No. 2303 liberally to both sides of washer, stud threads, and nut face.
2. Record free length on the four studs selected in crankcase machining.
3. Tighten all nuts in two steps as follows:
 - a. Apply torque to produce 11 ± 5 mils stud stretch on 567 and 645 engines (12 ± 5 mils on 710 engines) or 475 N·m (350 ft-lbs.) Ref. torque.
 - b. Apply torque to produce 25 ± 5 mils stud stretch on 567 and 645 engines (28 ± 5 mils on 710 engines) or 1017 N·m (750 ft-lbs.) Ref. torque.
4. Measure and record length of the four selected studs.
5. Remeasure steel bore diameters. The readings obtained must fall within the cap reapplication limits shown in Appendix Table 1. Record the vertical diameters in the appropriate columns on the Computation Worksheet.

SECTION III

QUALIFICATION OF LINE BORE

The preferred method requires the use of a precision granite table of sufficient size to support a bare crankcase. Upon request, the Electro-Motive Division Service Department will provide information concerning suppliers of precision granite tables.

GRANITE TABLE METHOD

The crankcase should be placed right side up on the precision granite table as shown in Fig. 4. It must be supported on its base rails as follows:

20 cylinder -	Supports centered 28" (711 mm) from ends of crankcase
16, 12, 8 and 6 cylinder -	Supports centered 12" (305 mm) from ends of crankcase

Preferably, the crankcase should be leveled so that dial indicator readings at the top of the two end MFMs are the same. However, a difference of within .010" (.25 mm) will not adversely affect results. In either case, set dial indicator to zero for reading at accessory end of No. 1 main bearing bore.

The granite table's sole purpose is to provide the source of reference for determining vertical bore alignment (case line data). This data, along with the recap steel bore diameter readings (Section II, E 5) is used to mathematically compute several alignment functions.

The case line readings, Fig. 5, are obtained using a dial indicator mounted on a surface gauge, 1/2" (12.7 mm) in from each main frame accessory and generator end face and must be made with all caps applied and torqued per Section II, E above. Take two independent sets of dial indicator readings. Any discrepancies between individual readings which are in excess of .0005" (0.013 mm) must be resolved. Record the final corrected readings on the computation worksheet for Granite Table Inspection (sample provided in the Appendix Section).

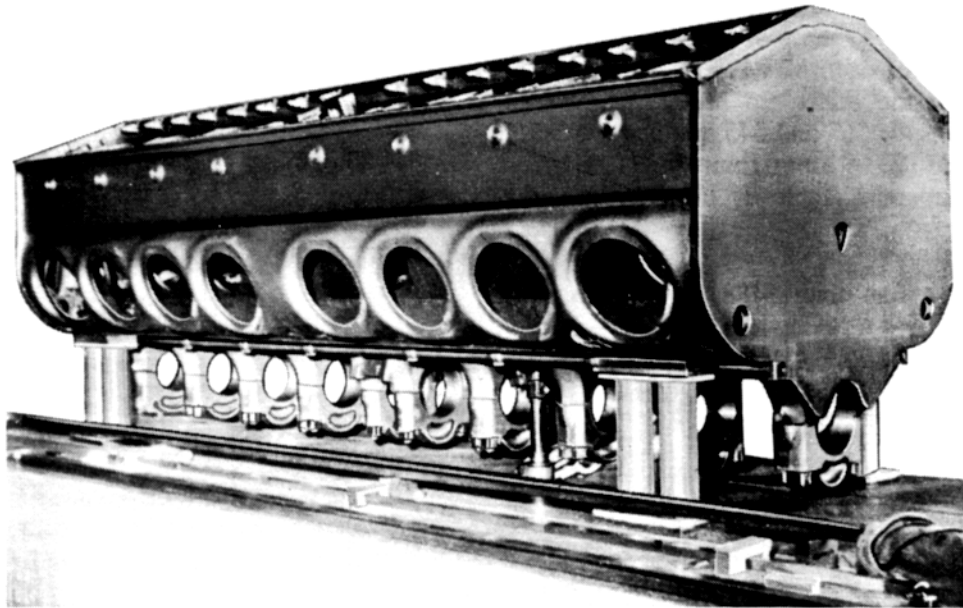
Before mathematically computing the alignment functions on the worksheet it is very important to understand each of the functions from a physical standpoint. The following definitions are provided along with an appropriate figure to explain each alignment concept.

1. Case Line —

The relative vertical position of the crankcase portion of the main bearing bores, Fig. 7.

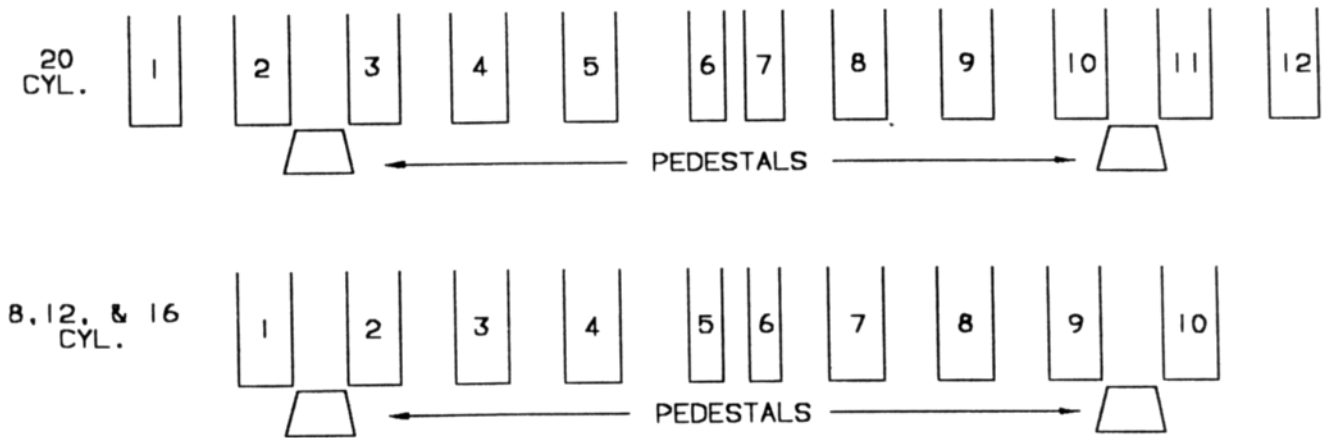
2. Cap Line —

The relative vertical position of the main bearing cap portion of the main bearing bores, Fig. 7.



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Fig. 4 - Crankcase Supported On Precision Granite Table



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Fig. 5 - Location Of Supports For Crankcase On Granite Table

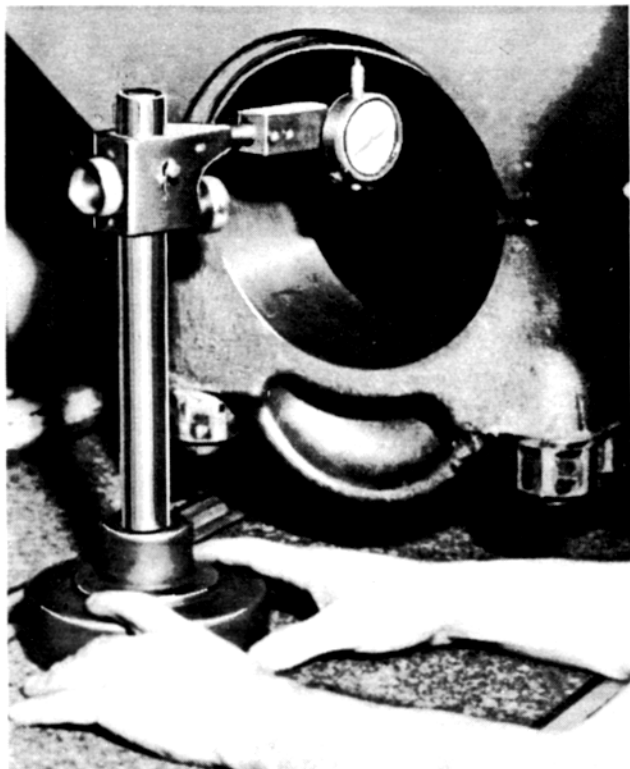


Fig. 6 - Bore Alignment Measurement
Using Dial Indicator

3. Vertical Bow —

The maximum deviation of the center of any one main bearing bore from a line drawn through the centers of the generator and accessory end main bearing bores is the crankcase vertical bow. The 16 cylinder crankcase shown in Fig. 8 has a maximum bow of $-.003''$ (-0.08 mm) or -3 mils sag. Positive bow is referred to as crown.

4. Step/Effective Step —

The difference in vertical position between two adjacent main bearing bores (case or cap line), Fig. 9.

5. Slope Change —

The relative vertical position of two adjacent effective steps (case or cap line) Figs. 10 and 11.

6. Case And Cap Tilt —

The difference between the accessory and generator positions of any one main bearing bore, Fig. 12.

An explanation on how to use the computation worksheet (for use with the granite table) follows. A master worksheet, as well as a completed example sheet is provided in the Appendix Section.

NOTE

For ease in calculating data, record as mils; where $.001''$ (0.03 mm) equals 1.0 mil.

1. Column A indicates the main bearing position number for 16 and 20 cylinder engines. The example sheet provided is for a 16 cylinder crankcase.
2. Columns B and C are case line dial indicator readings taken at the accessory and generator end respectively of each main bearing bore. The following sign convention must be used: Towards topdeck is pos. (+), away from topdeck is neg. (-).

NOTE

Dial indicator readings are recorded as positive (+) when gauge travel is towards the topdeck and negative (-) when gauge travel is away from the topdeck. Because of the inverted position of the gauge dial indicator, plus (+) gauge readings are recorded as minus values.

3. Column D is the average ordinate for case line for each main bearing bore. This is found by calculating the average of column B and C, or $[\text{Acc.} + \text{Gen.}]/2 = \text{AVG.}$

Example: $[(-2.5) + (-1.2)]/2 = -1.8$ mils.

4. Column E is the STEP between two adjacent main bearing bore averages. Step is calculated algebraically to determine sign convention using the following equation:

$$Y_2 - Y_1 = \text{Step } 1,2 \quad Y_3 - Y_2 = \text{Step } 2,3 \text{ etc.}$$

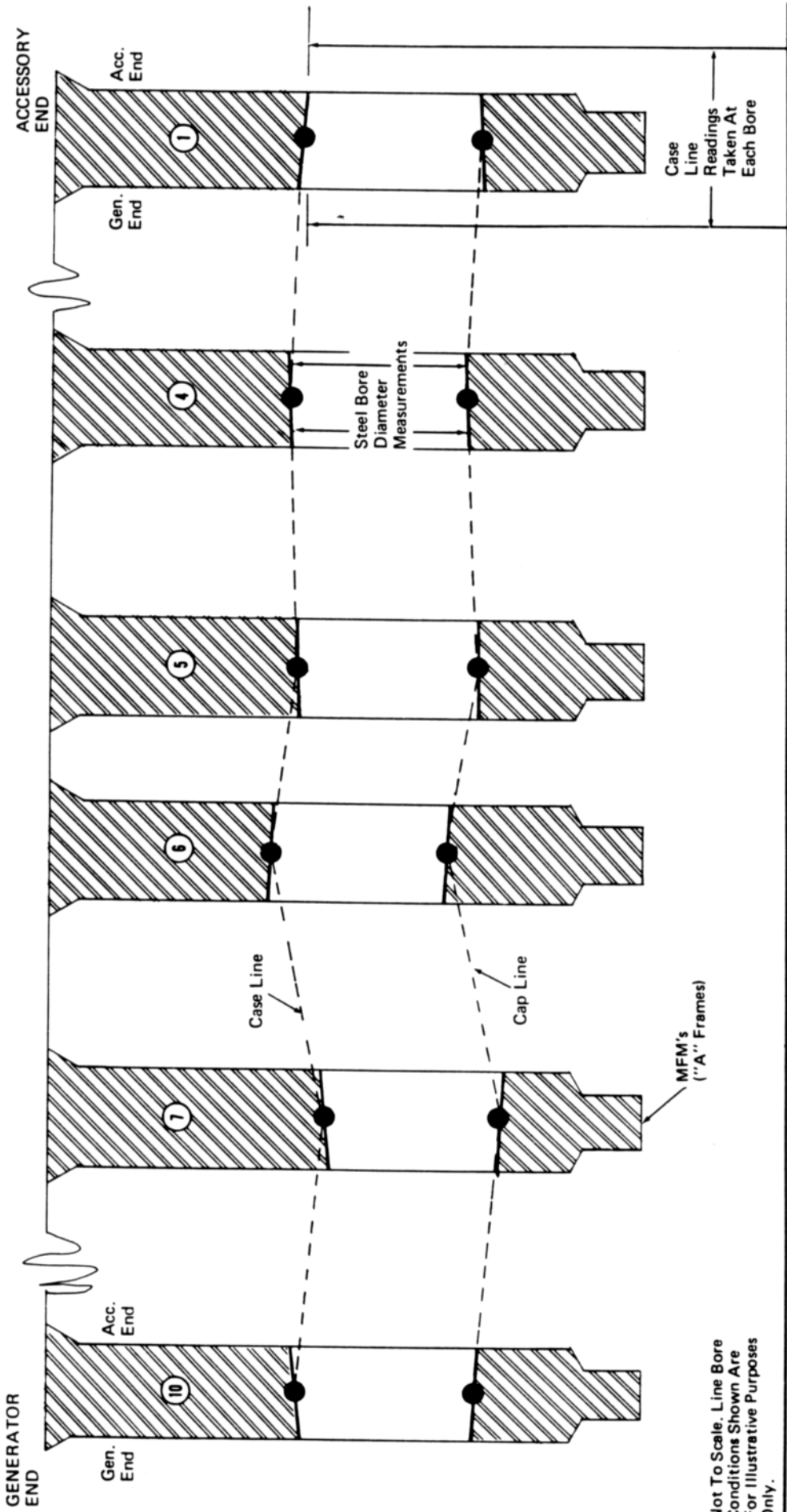
Example: $(-1.5) - (-1.8) = +0.3,$
 $(-0.3) - (-1.5) = +1.2$ etc.

5. Column F is a constant factor used to adjust step when the span between main bearing bores is not $16.625''$ (422.28 mm) for the purpose of calculating EFFECTIVE STEP. This is only necessary on 16 and 20 cyl. engines because the span distance between the two center main frames is $1/2$ the distance between any two intermediate main frames, therefore, the factor is 2.

6. Column G is the effective step which is the product (x's) of Column E and its corresponding factor in Column F.

7. Column H is the case line SLOPE CHANGE or the algebraic difference between two adjacent effective steps (Column G).

Example: $x_2 - x_1$ or $(+1.2)(+0.3) = +0.9,$
 $(+0.6) - (+1.2) = 0.6$ etc.



Not To Scale. Line Bore Conditions Shown Are For Illustrative Purposes Only.

GRANITE TABLE

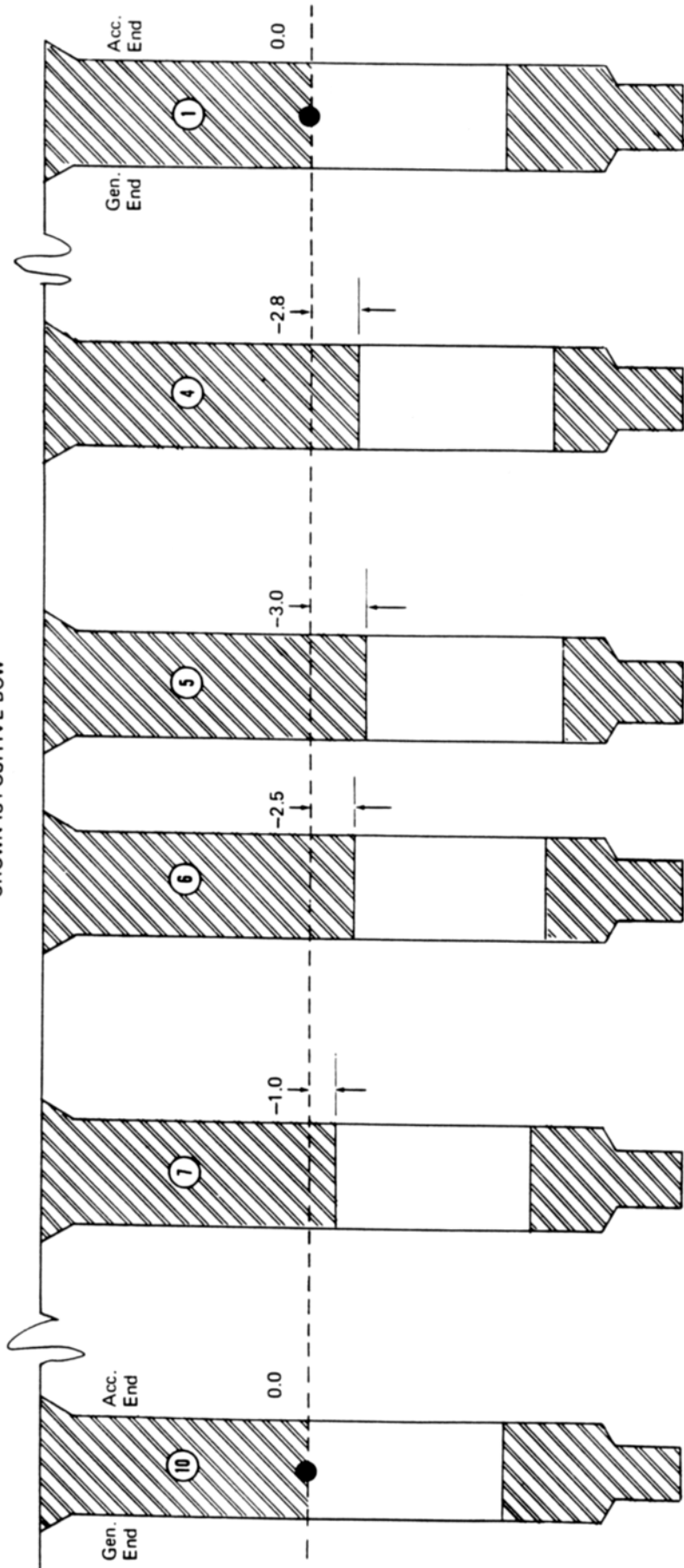
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Fig. 7 - Relative Vertical Position Of Case Line And Cap Line

NOTE

Single Dot • on top of bore represents average of two measurements taken 1/2" from end faces while dot on bottom of bore represents average of two calculations.

SAG IS NEGATIVE BOW
CROWN IS POSITIVE BOW



Not To Scale. Line Bore
Conditions Shown Are
For Illustrative Purposes
Only.

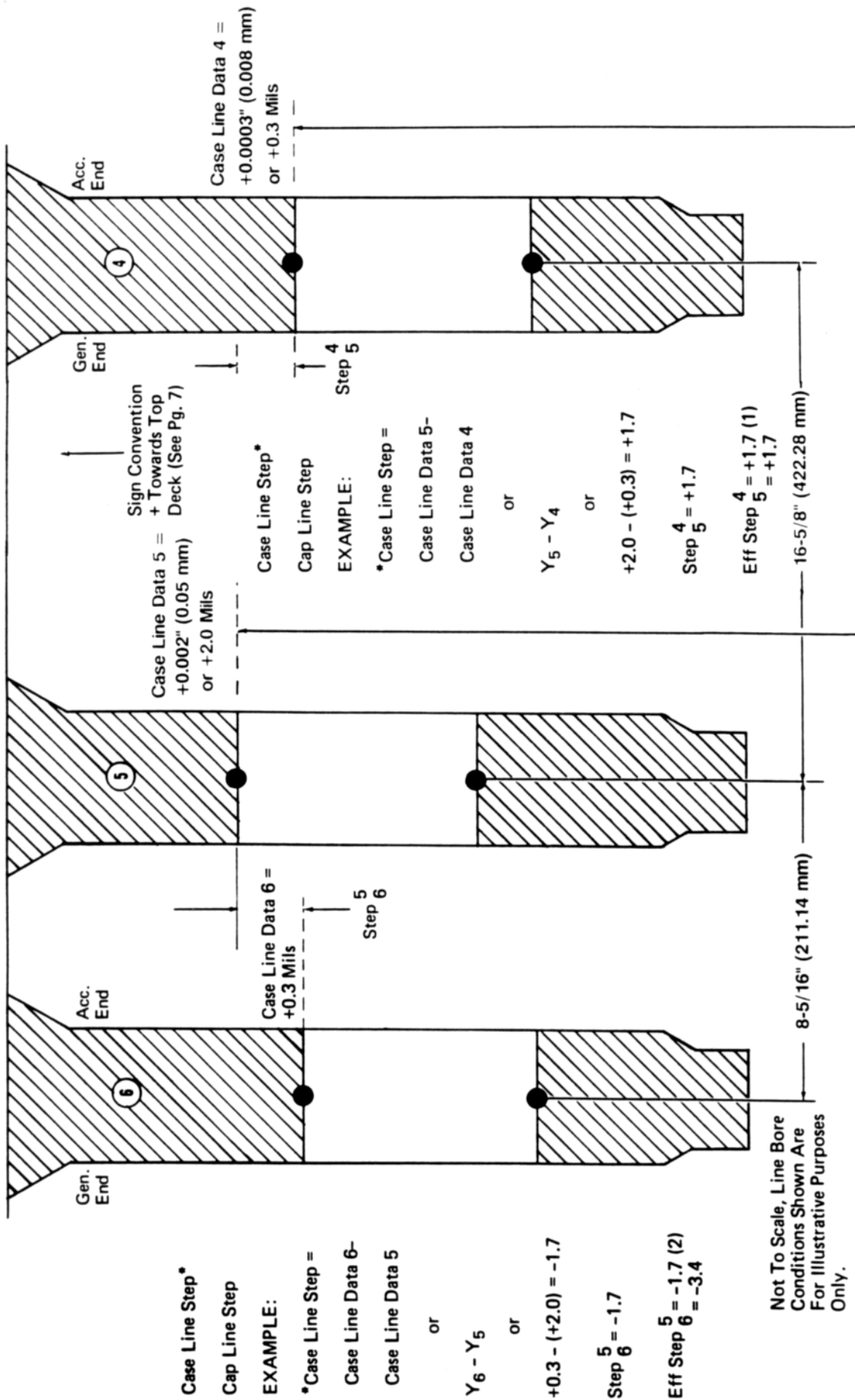
GRANITE TABLE

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Fig. 8 - Vertical Bow

NOTE

Single Dot • represents average of measurements 1/2" from outside of end faces.



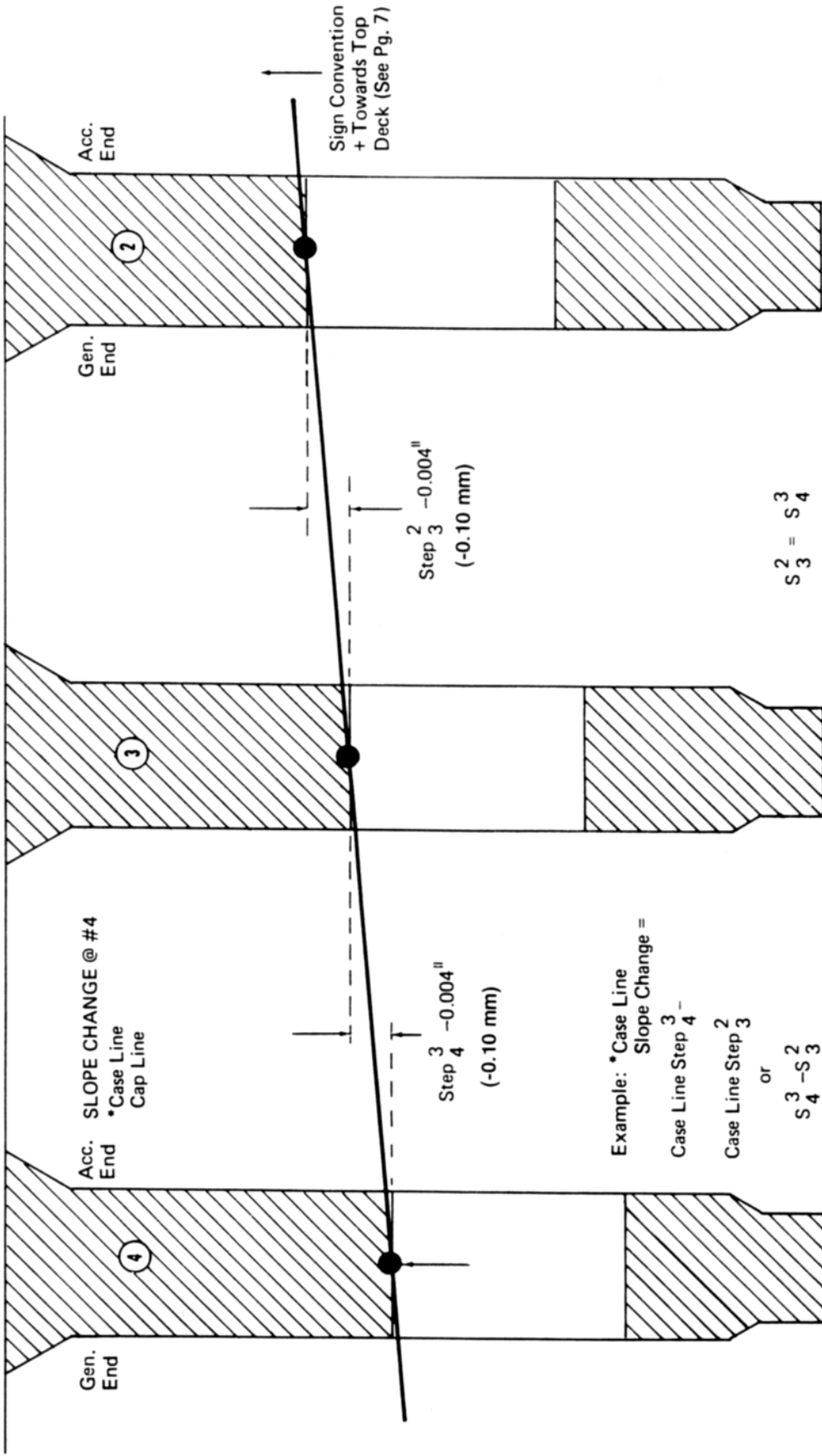
GRANITE TABLE

Fig. 9 - Step And Effective Step

NOTE

Single Dot • on top of bore represents average of two measurements taken 1/2" from end faces while dot on bottom of bore is the average of two calculations.

THE SLOPE CHANGE IS THE RELATIVE VERTICAL POSITION OF TWO ADJACENT EFFECTIVE STEPS.



Not To Scale. Line Bore Conditions Shown Are For Illustrative Purposes Only.

GRANITE TABLE

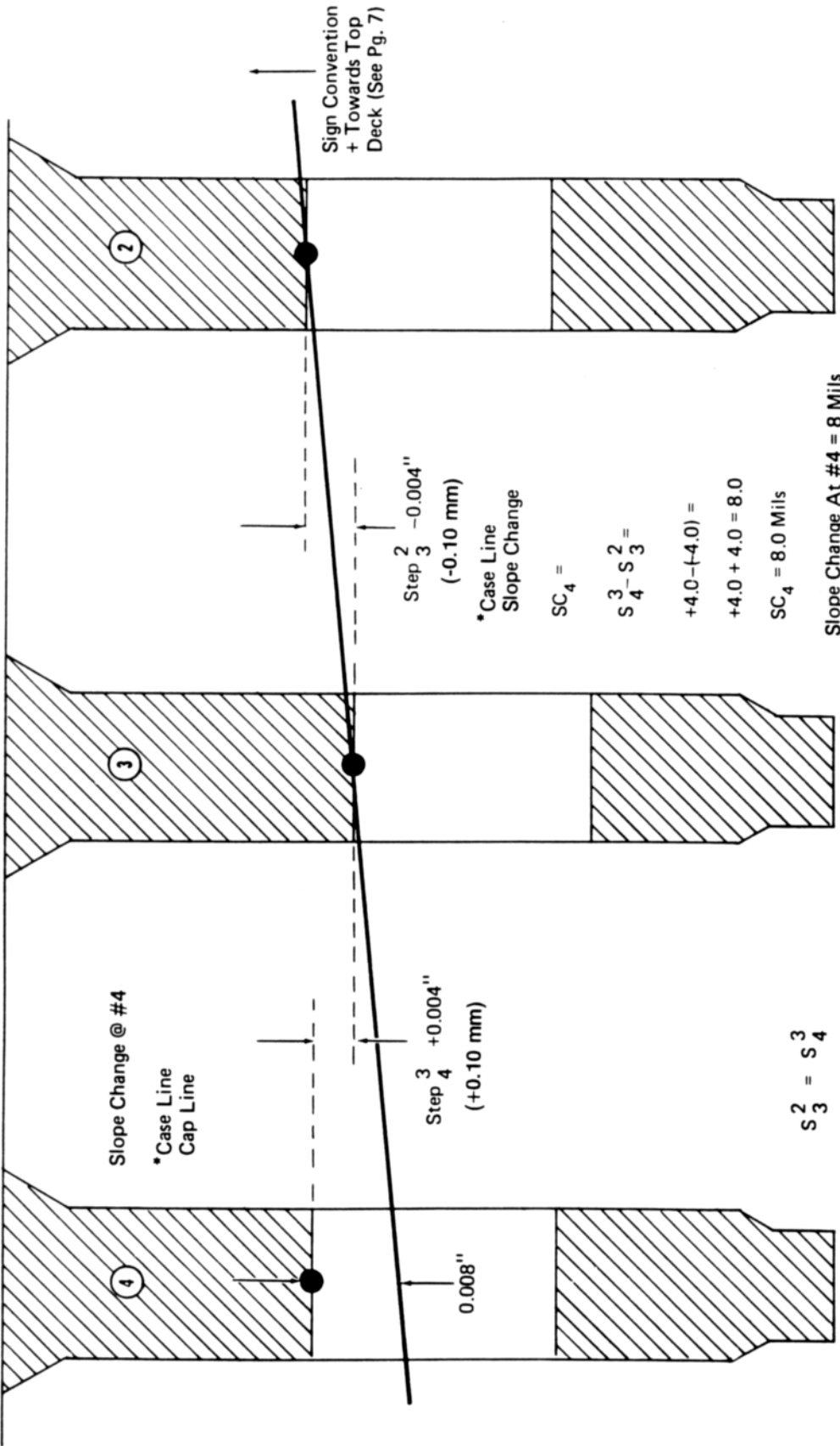
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Fig. 10 - Slope Change Vs. Step (Zero Slope Change Illustrated)

NOTE

Single Dot • represents average of measurements 1/2 from outside end faces.

SLOPE CHANGE IS THE RELATIVE VERTICAL POSITION OF TWO ADJACENT EFFECTIVE STEPS.



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GRANITE TABLE

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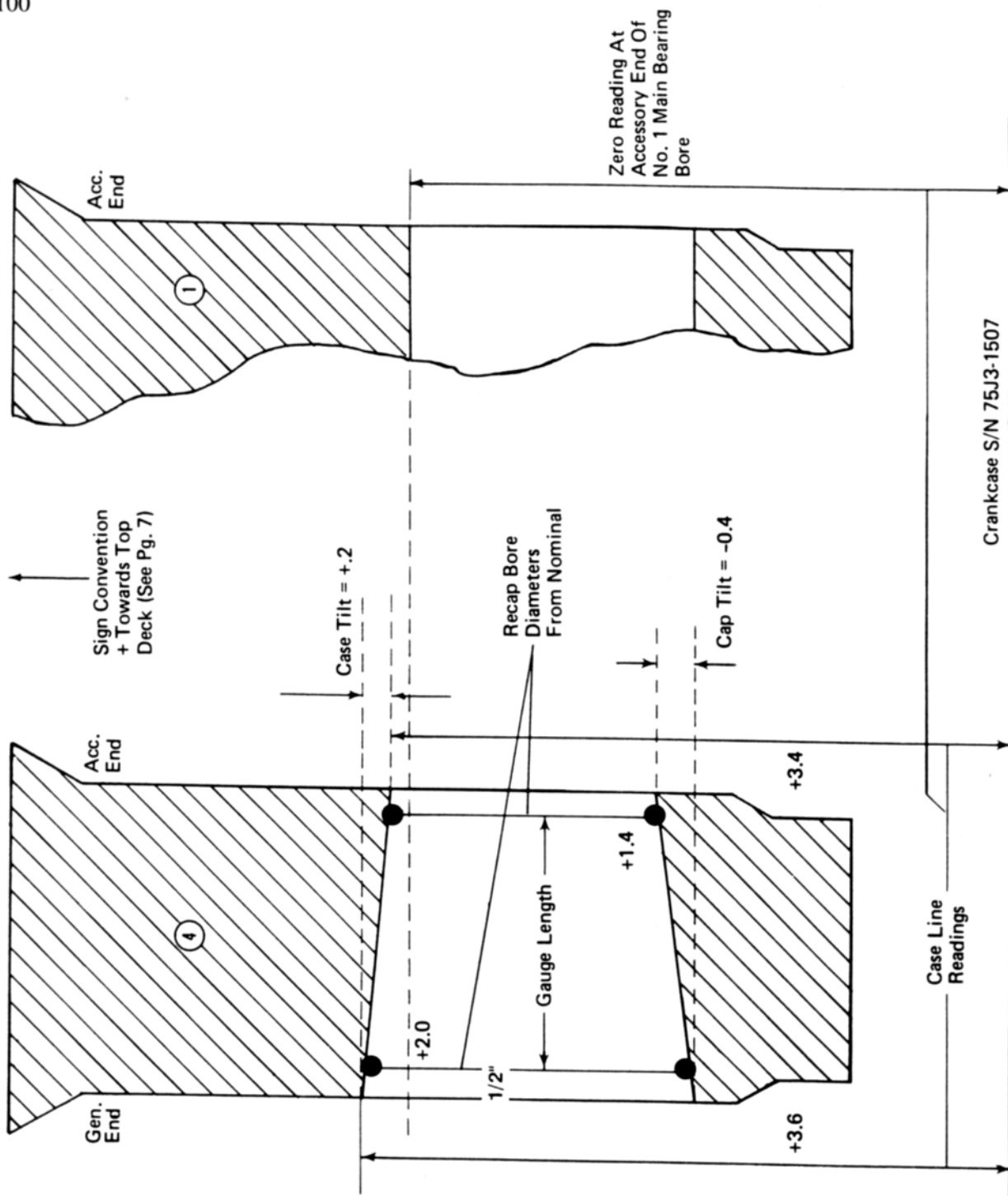
Fig. 11 - Slope Change Vs. Step (8 Mills Change Illustrated)

NOTE

Single Dot • represents average of measurements 1/2" from outside end faces.

8. Column I is the algebraic difference between the case line readings at the generator end (Column C) and the case line readings at the accessory end (Column B) which equals case line tilt (per gauge length), or $Z_G - Z_A$ or $(-1.2) - (-2.5) = +1.3$.
9. Columns J and K are vertical recap steel bore variations from the nominal bore diameter of 8.2500" (209.550mm) on 567 and 645 engines or 9.2500" (234.950mm) on 710 engines, taken at the accessory end and generator end respectively. The following sign convention is used: pos. (+) = Larger than nominal and neg. (-) = Smaller than nominal. Example: A typical vertical steel bore dia. of 8.2511" (209.578mm) is expressed +1.1 mils; 8.2488" (209.520mm) is -1.2 mils.
10. Column L is the steel bore difference, which is the accessory end steel bore size minus the generator end steel bore size.
Example: $(+1.1) - (+1.3) = -0.2$.
11. Column M is the CAP TILT per gauge length. CAP TILT is also equal to the difference of column N from column O or cap line calculations generator end minus cap line calculations accessory end. Example: $(-2.5) - (-3.6) = +1.1$. It is equal to the algebraic sum of Columns I and L or case line tilt plus steel bore difference. Example: $(+1.3) + (-0.2) = +1.1$.
12. Columns N and O are the cap line calculations for the accessory end and generator end respectively. It is the algebraic difference between the case line dial indicator readings and the steel bore variation from nominal; or Column B - Column J = Column N etc. and Column C - Column K = Column O etc.
Example: $(-2.5) - (+1.1) = -3.6$ and
 $(-1.2) - (+1.3) = -2.5$ etc.
13. Column P is the average ordinate for cap line at each main bearing bore and is calculated the same as the average ordinate for case line (Column D).
14. Columns Q through T are used to calculate CAP LINE step, effective step and slope change. The calculations are identical to those used to find case line, step, effective step and slope change respectively in Columns E, F, G, and H.

The main bearing line bore graph (master and example sheets provided in Appendix) is used to determine vertical bow. The numerical values listed in Columns D or P on the worksheet are used as the data points on the graph. A scale must be assigned to the horizontal lines on the graph so that they include the range of values from either Column D or P. Plot the data points on the graph under the appropriate main bearing position number. Draw a straight line to connect the two end data points and from the line of reference determine the max. vertical sag (-) or crown (+).



Case Tilt =
 Case Line Data, Gen. -
 Case Line Data, Acc.
 or
 $ZG - ZA$ or
 $3.6 - 3.4 = +0.2$

Cap Tilt =
 Case Tilt + *Steel Bore Diff.
 or
 $+0.2 + (-.6) = -0.4$

*Steel Bore Difference =
 Acc. End Steel Bore - Gen.
 End Steel Bore
 or
 $+1.4 - (+2.0) = -0.6$

NOTE

Measurements on top of bore to be taken 1/2" from outside of end faces. Dots on bottom represent calculated points.

Not To Scale. Line Bore Conditions Shown Are For Illustrative Purposes Only.

GRANITE TABLE

Crankcase S/N 75J3-1507

Fig. 12 - Case And Cap Tilt

SECTION IV

ENGINE REBUILD AND LOAD TESTING

Position the crankcase in such a manner as to facilitate main bearing cap removal and crankshaft installation. Apply new main bearings and the crankshaft in accordance with the procedure outlined in the appropriate Engine Maintenance Manual. Tighten all main bearing cap nuts per Section II, Item C 4. Assemble the

engine with new or requalified material and install in an appropriate test cell facility (engine application will so dictate) and prepare for break-in or load test. Be sure to properly fill the engine's lube oil system and pre-lube prior to start up.

It is recommended that the break-in or load test procedure used, includes a "feel over" check of the crankshaft/main bearing system before advancing engine speed and load.

APPENDIX

NOTE

The following limits apply only when the main bearing stud stretch is .025" ± .006" (0.64 ± 0.15mm) for 645 Engines and .028" ± .006" for 710 Engines.

**TABLE 1
STEEL BORE DIAMETER LIMITS
(DIMENSIONS IN INCHES)**

	REMACHINED				REUSE WITHOUT ANY BORE REMACHINING**	
	AS LINE BORED*		AFTER CAP RE-APPLICATION			
	567/645	710	567/645	710	567/645	710
Any One Diameter (Min.) (Max.)	8.2490 8.2510	9.2490 9.2510	8.2485 8.2515	9.2485 9.2515	8.2475 8.2535	9.2475 9.2535
Max. variation between accessory end diameter and gen. end diameter in any one steel bore	NO REQUIREMENT		±.002		±.003	
Cap Tilt (8, 12, 16 Cyl).	NO REQUIREMENT		±.002		±.003	
Cap Tilt caps 1 & 2 (20 Cyl.) caps 11 & 12 all others	NO REQUIREMENT		±.002		-.003 to +.001 -.001 to +.003 -.002 to +.002	

*Reference only. Dimensions obtained after cap reapplication are to be used for qualification.

**Exposed bronze on edges of main bearing requires remachining regardless of cap tilt.

NOTE

The data for 20 cylinder engines reuse without remachining is for crankcases with .011 to .020 sag. The tilt specifications are obtained by measuring **from the accessory end to the generator end**. Sag of less than .011 allows for a cap tilt of ±.003.

**TABLE 2
VERTICAL LINE BORE ALIGNMENT LIMITS**
(DIMENSIONS IN INCHES)**

CASE LINE & OR CAP LINE		REMACHINED		REUSE W/O REMACHINING
		MAX. BOW (-) OR CROWN (+)		MAX. BOW
20 cyl.		-.011	+.016	.020
16 cyl.		-.006	+.010	.012
12 cyl.		-.004	+.007	.008
8 cyl.		-.004	+.007	.008
6 cyl.		-.004	+.007	.008
Max. slope change - Thousandths/16.625"	Case Line	±.004*		±.006*
	Cap Line	±.005		±.008

*Reference only. Cap line dimensions are to be used for qualification.

**Also applies for horizontal alignment.

Main Bearing Number	ENGINE SERIAL NO.												CRANKCASE SERIAL NO.												DATE	
	Case Line Indicator Readings Acc. End	Case Line Indicator Readings Gen. End	Average (Acc. & Gen.) ²	Step, Case Line	Factor	Effective Step, Case Line	Slope Change, Case Line	Case Tilt	Bore Variation From Normal, Acc. End	Bore Variation From Normal, Gen. End	Steel Bore Difference, (Acc. - Gen.)	Cap Tilt	Cap Line Calculations, Acc. End	Cap Line Calculations, Gen. End	Average (Acc. - Gen.) ²	Step, Case Line	Factor	Effective Step, Case Line	Slope Change, Cap Line							
1					1												1									
2					1												1									
3					1												1									
4					1												1									
5					1												1									
6					2												2									
7					1												1									
8					1												1									
9					1												1									
10					1												1									
11					1												1									
12					1												1									
A	B	C	D	E	F	G	H	I	J	K	L	M	N	O	P	Q	R	S	T							
16 Cyl.																										
20 Cyl.																										

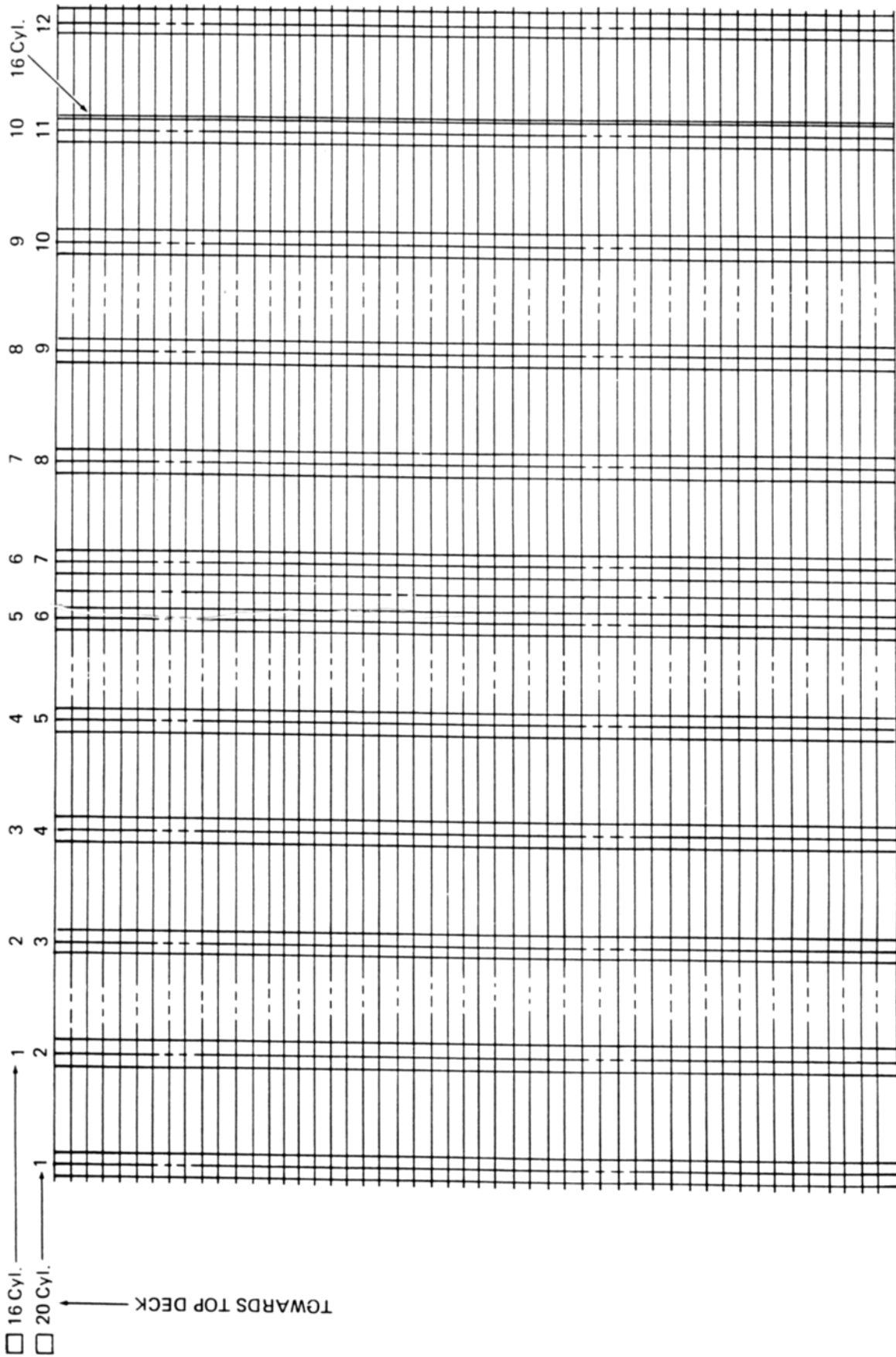
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Fig. A-1 - 16 & 20 Cylinder Crankcase Line Bore Computation Worksheet For Use With The Granite Table

Main Bearing Number	Case Line Readings Indicator	Case Line Acc. End	Case Line Indicator Readings	Average (Acc. & Gen.) ²	Step, Case Line	Factor	Effective Step, Case Line	Slope Change, Case Line	Case Tilt	Bore Variation From Normal, Acc. End	Bore Variation From Normal, Gen. End	Steel Bore Difference, (Acc. - Gen.)	Cap Tilt	Cap Line Calculations, Acc. End	Cap Line Calculations, Gen. End	Average (Acc. - Gen.) ²	Step, Case Line	Factor	Effective Step, Case Line	Slope Change, Case Line	
1						1												1			
2	-2.5		-1.2	-1.8	.3	1	.3	.9	1.3	1.1	1.3	-.2	1.1	-3.6	-2.5	-3.0	2.4	1	2.4		
3	-1.7		-1.3	-1.5	1.2	1	1.2	-.6	.4	-.6	-1.2	.6	1.0	-1.1	-.1	-.6	.1	1	.1		-2.3
4	-6		-.1	-.3	.6	1	.6	-.6	.5	.3	0.0	.3	.8	-.9	-.1	-.5	1.9	1	1.9		1.8
5	0.0		.5	.3	1.7	1	1.7	1.1	.5	-1.1	-1.2	.1	.6	1.1	1.7	1.4	-.2	1	-.2		-2.0
6	2.0		2.0	2.0	.6	2	1.2	-.5	0.0	.9	.6	.3	.3	1.1	1.4	1.2	.7	2	1.4		1.6
7	2.8		2.4	2.6	2.1	1	2.1	.9	-.4	.5	.9	-.4	-.8	2.3	1.5	1.9	3.2	1	3.2		1.8
8	5.4		4.1	4.7	-1.7	1	-1.7	-3.8	-1.3	-.5	-.2	-.3	-1.6	5.9	4.3	5.1	-2.2	1	-2.2		-5.4
9	2.8		3.3	3.0	-6	1	-.6	1.1	.5	-.2	.5	-.7	-.2	3.0	2.8	2.9	-1.4	1	-1.4		.8
10	2.4		2.4	2.4	-2.1	1	-2.1	-1.5	0.0	1.0	.9	.1	.1	1.4	1.5	1.5	-1.9	1	-1.9		-.5
11	.5		0.0	.3		1			-.5	.7	.7	0.0	-.5	-.2	-.7	-.4		1			
12																					
A	B	C	D	E	F	G	H	I	J	K	L	M	N	O	P	Q	R	S	T		
16 Cyl																					
20 Cyl																					

22263

Fig. A-2 - 16 & 20 Cylinder Crankcase Line Bore Computation Worksheet For Use With The Granite Table



Q ENG.

ENG. SERIAL NO. _____

C'CASE SERIAL NO. _____

DATE _____

INSPECTOR _____

Fig. A-3 - Main Bearing Line Bore Graph

ENG. SERIAL NO. 76C3-1518
 INSPECTOR _____
 MODEL SD45

CASE SERIAL NO. 76A-115
 DATE _____
 TYPE 20-645E3

A	B	C	D	E	F	G	H	I	J	K
20 Cyl. Run No. 16 Cyl.	20 Cyl. Bearing No. 16 Cyl.	Cap Line Dial Indicator Rdgs. Acc. End	Cap Line Dial Indicator Rdgs. Gen. End	Average (A + G) / 2	Step	Factor (Constant)	Effective Step	Uncorrected Slope Change	Bar Correction	Corrected Slope Change
1	1	4.7	5.5	5.1	-1.1	1	-1.1			
	2	3.7	4.4	4.0	.3	1	.3	1.4	-.6	.8
	3	3.6	5.0	4.3						
2 1	2 1	4.3	5.0	4.6	-4	1	-4	.4	-.6	-.2
	3 2	3.7	4.8	4.2	0.0	1	0.0			
	4 3	4.1	4.3	4.2						
3 2	3 2	3.9	5.0	4.4	-.3	1	-.3	0.0	-.6	-.6
	4 3	4.1	4.1	4.1	-.3	1	-.3			
	5 4	3.8	3.9	3.8						
4 3	4 3	4.5	4.7	4.6	-.6	1	-.6	1.5	-.6	.9
	5 4	4.3	3.8	4.0	.9	1	.9			
	6 5	4.7	5.2	4.9						
5 4	5 4	4.1	3.7	3.9	1.0	1	1.0	-1.6	-.6	-2.2
	6 5	4.7	5.2	4.9	-.3	2	-.6			
	7 6	4.5	4.7	4.6						
6 5	6 5	5.0	5.4	5.2	-.3	2	-.6	-.3	-.6	-.9
	7 6	4.9	5.0	4.9	-.9	1	-.9			
	8 7	3.2	4.4	3.8						
7 6	7 6	4.7	4.9	4.8	-.9	1	-.9	1.4	-.6	.8
	8 7	3.3	4.5	3.9	.5	1	.5			
	9 8	4.1	4.8	4.4						
8 7	8 7	3.6	4.6	4.1	.6	1	.6	-.7	-.6	-1.3
	9 8	4.4	5.0	4.7	-.1	1	-.1			
	10 9	3.7	5.5	4.6						
9 8	9 8	4.9	5.6	5.2	-.5	1	-.5	-.6	-.6	-1.2
	10 9	4.1	5.4	4.7	-1.1	1	-1.1			
	11 10	3.4	3.9	3.6						
10	10	3.9	5.7	4.8	-.9	1	-.9	1.8	-.6	1.2
	11	3.6	4.2	3.9	.9	1	.9			
	12	4.9	4.8	4.8						

Fig. A-4 - 16 & 20 Cylinder Line Bore Computation Worksheet For Use With Granite Measuring Bar 8467738 (Sheet 1 of 2)

ENGINE SERIAL NO. 76C3-1518

CASE SERIAL NO. 76A-115

INSPECTOR _____

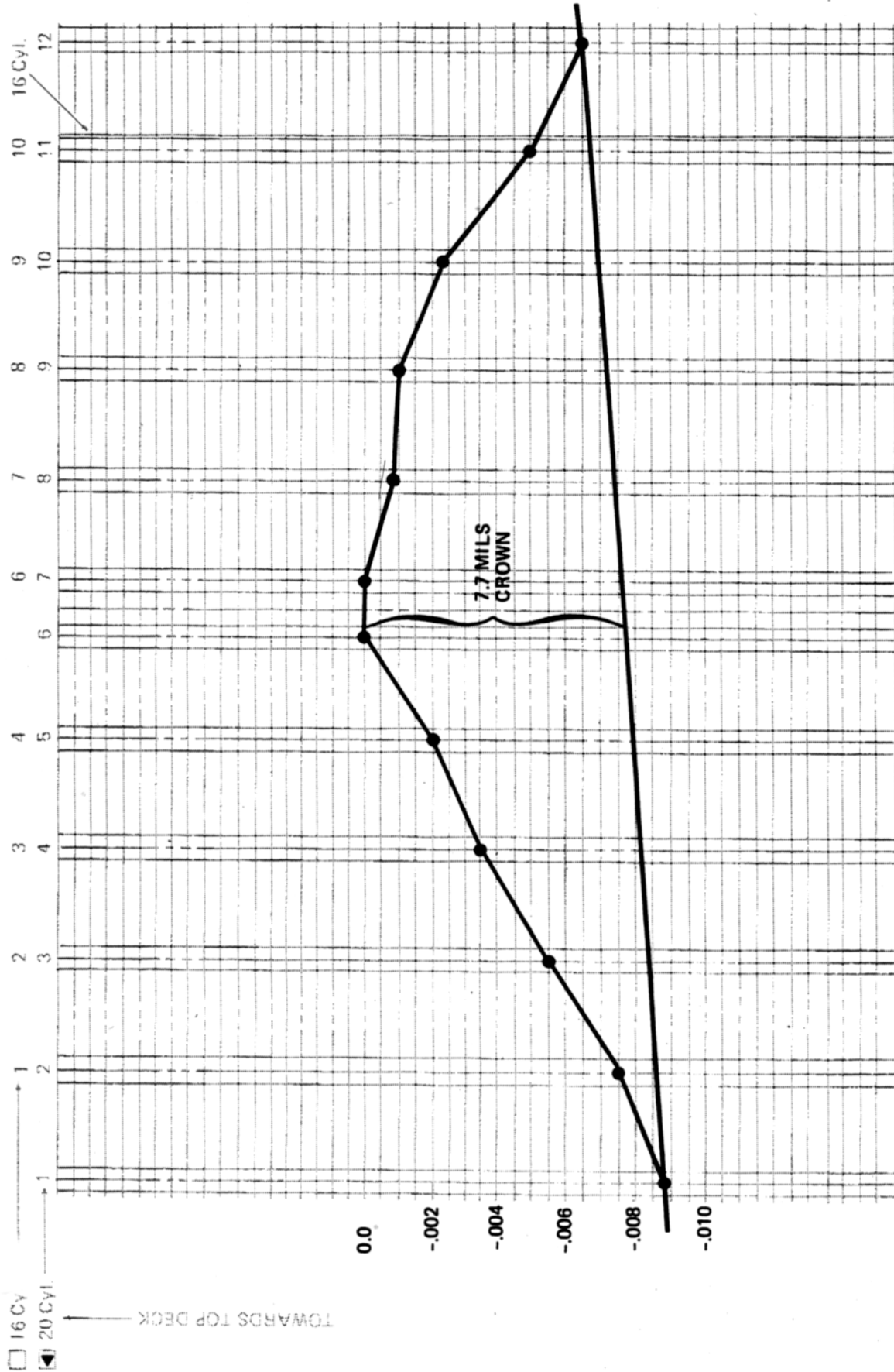
DATE _____

MODEL SD45

TYPE 20-645E3

		L	M	N	O	P
		20 Cyl. Bearing No. / 16 Cyl.	Slope Change Corrected	Cum. Sum of Change From Step I	Cum. Sum Of Change From Step II	Vertical Bow/Crown Data (From Line Bore Graph, Fig. A5)
1				-8.8	0.0	
2	1	.8	-1.3	-7.5	1.1	
3	2	-.2	-2.1	-5.4	3.0	
4	3	-.6	-1.9	-3.5	4.7	
5	4	.9	-1.3	-2.2	5.6	
6	5	-2.2	-2.2	0.0	7.7	
7	6	-.9		0.0	7.6	ENGINE Q
8	7	.8	-.9	-.9	6.5	
9	8	-1.3	-.1	-1.0	6.1	
10	9	-1.2	-1.4	-2.4	4.4	
11	10	1.2	-2.6	-5.0	1.7	
12			-1.4	-6.4	0.0	

Fig. A-4 - 16 & 20 Cylinder Line Bore Computation Worksheet For Use With Granite Measuring Bar 8467738 (Sheet 2 of 2)



ENG. SERIAL NO. 76C3-1518 DATE _____
 C'CASE SERIAL NO. 76A-115 INSPECTOR _____

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Fig. A-5 - Main Bearing Line Bore Graph — 20 Cyl.

SERVICE DATA

EQUIPMENT LIST

	Part No.
Dial Bore Gauge	8275258
Master (8.2500" Nominal)	9321276
Master (9.2500" Nominal)	40038306
 Serration Gauge — Includes Master	
567 & 645 Engines	8177167
710 Engines	40038647
 Precrush Plates	8488128
 Texaco Threadtex Lubricant #2303 (approx. 18.93 liter [5 gal.])	8307731
 Granite Inspection Table	Contact EMD Service Department For Supplier Information
 Support Blocks	File Drawing #778
 Dial Indicator Gauge — Case Line Readings - Table Method	File Drawing #891
 Dial Indicator Gauge	8467737

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